

## (12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(19) World Intellectual Property Organization International Bureau



# - I TORIO PILITURI CORRE CIRIO ROBIN COMO CON ENTRE ELIMPRICA DE LA CORRECCIONE COMO COMO COMO COMO COMO COMO C

(43) International Publication Date 18 March 2004 (18.03.2004)

PCT

(10) International Publication Number WO 2004/022856 A1

(51) International Patent Classification7:

\_

E02B 9/08

(21) International Application Number:

PCT/GB2003/003845

(22) International Filing Date:

5 September 2003 (05.09.2003)

(25) Filing Language:

English

(26) Publication Language:

English

(30) Priority Data: 0220626.6

5 Can

5 September 2002 (05.09.2002) GE

- (71) Applicant (for all designated States except US): ROBERT GORDON UNIVERSITY [GB/GB]; Schoolhill, Aberdenn AB10 1FR (GB).
- (72) Inventors; and
- (75) Inventors/Applicants (for US only): OWEN, Alan [GB/GB]; Braeside Schoolhouse, Auchnagatt, Ellon AB41 8YE (GB). BRYDEN, Ian, Gordon [GB/GB]; 62 Brentfield Circle, Ellon, Aberdeenshire AB41 9DD (GB).
- (74) Agent: MURGITROYD & COMPANY; Scotland House, 165-169 Scotland Street, Glasgow G5 8PL (GB).

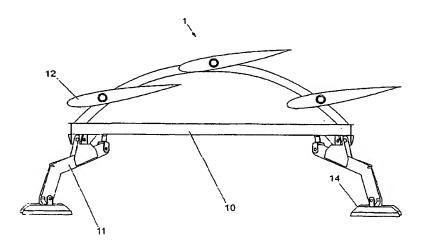
- (81) Designated States (national): AE, AG, AL, AM, AT, AU, AZ, BA, BB, BG, BR, BY, BZ, CA, CH, CN, CO, CR, CU, CZ, DE, DK, DM, DZ, EC, EE, ES, FI, GB, GD, GE, GH, GM, HR, HU, ID, IL, IN, IS, JP, KE, KG, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, MA, MD, MG, MK, MN, MW, MX, MZ, NI, NO, NZ, OM, PG, PH, PL, PT, RO, RU, SC, SD, SE, SG, SK, SL, SY, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, YU, ZA, ZM, ZW.
- (84) Designated States (regional): ARIPO patent (GH, GM, KE, LS, MW, MZ, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European patent (AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HU, IE, IT, LU, MC, NL, PT, RO, SE, SI, SK, TR), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

#### Published:

- with international search report
- before the expiration of the time limit for amending the claims and to be republished in the event of receipt of amendments

For two-letter codes and other abbreviations. refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

(54) Title: APPARATUS FOR CONTROLLING UNDERWATER BASED EQUIPMENT



(57) Abstract: The apparatus may include a space frame (10, 110) on which is mounted at least one hydrofoil (12, 112) for generating positive or negative lift. The frame (10, 110) is attachable to underwater equipment such as a turbine. The hydrofoils (12, 112) are adapted to produce negative lift when a flow of liquid passes over them and so in use cause the apparatus (10, 110) and attached equipment to sink to the seabed. The flow of water over the hydrofoils continue to produce negative lift and so maintain the apparatus (1, 100) on the seabed. In certain embodiments, the hydrofoils (12) can typically be set to a passive configuration in which they flip over when the current flow changes direction. Furthermore, the hydrofoils (12) are selectively rotatable to provide an angle of attack such that they may be adapted to provide positive lift when it is necessary to remove the apparatus (1) from the water.

WO 2004/022856

8/1919

"Apparatus	for	Controlling	Underwater	Based

2 Equipment"

3 4

1

## Technical field

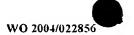
5

- 6 The invention relates to an underwater location
- 7 device such as may be used for controlling the
- 8 launch, positioning or recovery of a tidal turbine
- 9 or other underwater equipment. It should be noted
- 10 that the example of a tidal turbine is used herein
- 11 but the invention is not limited to such uses.

12 13

## Background art

- 15 Tidal currents offer a considerable source of
- 16 sustainable energy at various sites throughout the
- 17 world, usually within easy reach of land and in
- 18 relatively shallow waters. Tidal currents are
- 19 created by movement of the tides around the earth
- 20 producing a varying sea level, dependent on the
- 21 phases of the moon and sun. As the sea levels vary,
- 22 so the waters attempt to maintain equilibrium'

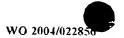




2

subject to gravitational forces, thus inducing flow 1 from one area of sea to another. This flow is . 5 modified by a number of factors such as, the 3 Coriolis forces due to the earth rotation, 4 earth/moon/sun alignment, local topography, 5 6 atmospheric pressure and temperature and salinity gradients. The major advantage of tidal power 7 8 generation is its regularity, which can be predicted 9 for years in advance. 10 According to a study by the ETSU (Energy Technology 11 Support Unit) the United Kingdom may obtain up to 20 12 percent of its total electricity by using these 13 14 systems to collect energy from fast moving tidal currents that exist in channels and offshore areas. 15 Similar resources have been noted to exist elsewhere 16 17 such as in the Straits of Messina, between Sicily 18 and mainland Italy. 19 The most powerful flows tend to occur in areas of 20 restriction, either by width or depth, but for the 21 22 same reasons are not suitable for widespread 23 exploitation by large, fixed devices which require a minimum rotor area, and therefore water depth, to 24 25 justify the costs of installation and maintenance. It is assumed from the outset that new tidal barrage 26 systems are unlikely ever to be pursued due to their 27 28 inherent properties of high cost, delayed financial return, and serious environmental consequences. 29

- 31 The considerable size of the available resource has
- 32 attracted various proposals for its exploitation.



32



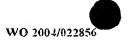
#### PCT/GB2003/003845

3

The following represents the existing systems within 1 the field of tidal current energy extraction. 2 assumed that power transmission problems will be 3 equal for any system, and that all systems will 4 require some form of non-toxic anti-fouling agent. 5 6 There also exist operational environmental impacts 7 common to all methods of tidal power generation, 8 such as, an inherent risk of collision damage to 9 fish and marine mammals, redirection of currents and 10 the sediments and food particles contained within 11 12 them, and shipping, particularly fishing. 13 A first type of tidal current energy extraction 14 system encountered on the market is the Monopile 15 16 This technology is well known and understood by contractors familiar with the offshore 17 18 oil industry. It consists of twin axial flow turbines, each turbine driving a generator via a 19 gearbox, mounted on streamlined cantilevers either 20 side of a circular section, vertical steel monopile. 21 It is anticipated that a number of structures will 22 be grouped together in 'farms'. The planning of 23 such a tidal 'farm' would need to be accurately 24 modelled for wake effects, as once installed, the 25 monopile is expensive to re-site. In addition, 26 operational depth is restricted to the 20m - 35m27 range. Concerning the installation and maintenance, 28 monopile systems require a hole to be drilled in 29 suitable bedrock and the base of the turbine tower 30

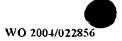
is secured within the socket so produced. Existing

monopile support mechanisms for presenting a tidal



4 turbine to the tidal currents are expensive, thus 1 making only a few sites economically viable for 2 3 power generation and requiring considerable sub sea 4 engineering expertise. 5 б The current monopile systems permit raising the 7 turbines above water level for maintenance and repair, which is beneficial, but the long-term (i.e. 8 20 years) reliability and corrosion resistance of 9 10 the necessary mechanism must be questionable. protrusion of the piles above sea level would reduce 11 the likelihood of impact with passing vessels. 12 13 Concerning the environmental and decommissioning 14 15 issues, the impact of installation would be considerable, especially to the benthic flora and 16 17 fauna, but subsequently the piles may become areas of shelter and therefore, populated. To minimise 18 19 the danger to shipping and fishing, decommissioning would require complete removal of the piles, which 20 21 would disturb the benthic population once again. 22 23 A second type of tidal current energy extraction system that exists in the prior art is the floating 24 25 tether. This floating tether device is anchored to the seabed with a mooring cable and suspended clear 26 27 of the seabed using a flotation buoy. The axial flow tidal current turbine is free to position 28 29 itself into the direction of the tidal flow, which

obviates the need for a yaw mechanism.



32



## PCT/GB2003/003845

5 Several prototypes have already been developed 1 including a 10-kilowatt device tested in Scotland in 2 1994. At present, the arrangement is unlikely to be 3 suitable for large power output installations due to 4 the relative sizes of anchor, turbine and float. 5 occasions of relatively high velocity tidal streams 6 7 (e.g. spring tides), if the anchor holds, the turbine will be dragged lower in the water with the 8 unwanted potential to collide with the seabed. 9 10 Concerning the installation of the floating tether 11 system, it is relatively quick and inexpensive. 12 However, visual inspection would need to be frequent 13 as the structure is likely to be subject to storm 14 damage and fatigue loading of the cable, leading to 15 possible loss of the supporting float and subsequent 16 sinking of the device, or loss of anchorage and 17 subsequent drifting. Once sunk, the device would be 18 open to damage by the oscillating tidal currents and 19 could prove difficult to recover, whilst a drifting 20 device would potentially cause damage to any other 21 22 moored turbines in its path. 23 Due to the length of tether required and the random 24 25 positioning of the device at any one time, this arrangement is not suitable for closely grouped 26 tidal farms and a safe spread would fail to make 27 28 economical use of the power available in a given 29 area. For the same reasons, this type of 30 arrangement would present a hazard to all forms of

shipping, large and small. It would, however

present a possible solution to a one-off, small



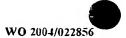
WO 2004/022856

PCT/GB2003/003845

6 scale installation in areas such as the mouth of a 1 sea loch. Concerning the environmental impacts of 2 installation and decommissioning of the floating 3 tether systems, it will be minimal, leaving no 4 footprint on removal. 5 6 A third type of tidal current energy extraction 7 system that also exists in the prior art is the 8 9 oscillating hydroplane system. In that system, a central post mounted on five legs supports a complex 10 mechanism comprising two interconnected symmetrical 11 hydrofoils. These hydrofoils are used to pump high-12 pressure oil, which drives an electrical generator 13 via a hydraulic motor. At the end of each stroke, 14 the hydrofoils are tilted to give the required angle 15 of attack to produce the return stroke, thus 16 creating an oscillating motion. 17 18 Concerning the installation and maintenance, at 19 present, the oscillating hydroplane system does not 20 yet possess a launch and recovery mechanism. As a 21 result of the constant oscillations and considerable 22 number of moving parts, it is probable that this 23 device will be subject to high dynamic loading and 24 subsequent fatigue stress. 25 The upward stroke of the hydrofoils will tend to lift the device off the 26 seabed and hence increase the possibility of it 27 being washed away at high tidal stream velocities. 28 Concerning the environmental impacts of installation

29

30 and decommissioning of the oscillating hydroplane 31 systems, they are expected to be minimal, leaving no 32



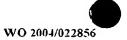
31

negative lift.

#### PCT/GB2003/003845

7

footprint on removal. 1 However, this cannot be 2 confirmed until a launch/recovery mechanism is 3 proposed. Using high pressure oil as a means of 4 power transmission does however introduce the 5 possibility of pollution in the event of leakage. 6 7 Some 'tidal' energy extraction systems can also be 8 used in freshwater applications such as rivers. 9 With these existing systems and designs, it is a 10 problem that their instabilities during operations 11 as well as during launch and recovery, if possible, 12 might cause damage. In addition, since these 13 systems are becoming larger and larger, the frequent 14 15 installation and maintenance operations will become more and more difficult and expensive. 16 17 18 Summary of the invention 19 20 It is an object of the present invention to obviate 21 or mitigate the problems of controlling underwater 22 equipment in a flowstream. 23 24 In a first aspect, the invention described herein relates to an apparatus for controlling underwater 25 26 equipment comprising: 27 attachment means for attaching underwater equipment to the apparatus; and 28 29 at least one member for generating positive or



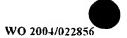


#### PCT/GB2003/003845

8

Preferably, the at least one member is adapted to 1 create a negative lift due to fluid flow in a first 2 direction and is adapted to create a negative lift 3 due to fluid flow in a second, different, direction. 4 5 6 Preferably, the first and second directions are 7 generally opposite to each other. 8 9 Preferably, the apparatus is adapted to anchor the 10 underwater equipment to a sea- or river-bed. 11 12 Preferably, the attachment means is adapted to attach the underwater equipment in close proximity 13 14 to the centre of gravity of the apparatus. 15 16 Preferably, the space frame is mounted on a number 17 of feet equipped with slippage prevention means, 18 which may be an arrangement of spikes or the like, to typically resist slipping by shear force rather 19 20 than relying on friction alone such that, in use, the negative lift will preferably tend to force said 21 22 slippage prevention means into a sea- or river-bed thus resisting the drag forces acting on the space 23 24 frame tangentially to the seabed. 25 26 . Preferably, the at least one member comprises at 27 least one hydrofoil. 28 29 Typically, differences in pressure acting on opposing surfaces of each of the at least one member 30

due to a predetermined angle of attack causes said





9 at least one member to generate negative or positive I 2 lift. 3 Preferably, the apparatus is adapted to control the 4 launch and/or recovery of the underwater equipment 5 6 attached to it. 7 . In a preferred embodiment, the at least one members 8 9 are rotatable to any position and even more preferably in the region of 160° to 200° about a 10 longitudinal axis of the respective member. 11 12 Preferably, the hydrofoils are symmetrical. 13 14 Said at least one members preferably comprise at 15 least one hydrofoils which are more preferably self-16 rectifying static hydrofoils, which may be capable 17 of passive rotation about an axis such that each 18 hydrofoil maintains alignment with a periodically 19 20 reciprocating rectilinear flow. 21 Moreover, the at least one members are preferably 22 moveable between a first configuration in which they 23 are capable of generating positive lift and a second 24 25 configuration in which they are capable of 26 generating negative lift.

27

Preferably, the at least one member has a variable 28 actuating means to vary the positive or negative 29 30 lift generated by the member.





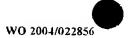
#### WO 2004/022856

#### PCT/GB2003/003845

10

- Preferably, said actuating means comprises a motor
- which may be a hydraulic, pneumatic or electric
- 3 actuated motor. Preferably, a shaft member is
- 4 actuated when a change between first and second
- 5 configurations is required, said actuation typically
- 6 causing the shaft member to rotate through a
- 7 predetermined angle, which may be in the region of
- 8 180°,

- 10 Preferably, said apparatus comprises a support
- 11 framework which is typically composed of sub
- 12 frameworks, where a number of shaft members are
- 13 connected to the framework and on which said
- 14 symmetrical hydrofoils are coupled. Preferably, the
- at least one hydrofoils are coupled to the support
- 16 framework by a respective bearing member connected
- 17 to the hydrofoil. The bearing member of the
- hydrofoil is typically coupled to the shaft member
- of the framework, the bearing member and shaft
- 20 member combining to provide a rotation enabling
- 21 portion and a rotation prevention portion.
- 22 Preferably, the bearing member is substantially
- 23 cylindrical. The rotation prevention portion
- 24 typically comprises at least one stop members (which
- 25 may be in the form of lugs mounted on the shaft
- 26 member) and which are adapted to engage with at
- 27 least one respective stop members (which may also be
- 28 lugs) mounted on the respective bearing member of
- each hydrofoil. Typically, the bearing member
- 30 comprises a pair of stop members which are spaced
- 31 apart around its inner circumference, typically
- 32 being spaced apart by approximately 180°.

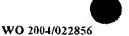


## PCT/GB2003/003845

11

Typically, the shaft member comprises a pair of stop 1 members which are spaced apart around its outer 2 circumference, typically being spaced apart by 3 approximately 180°. Preferably, one of the bearing 4 stop members is engageable with a respective shaft 5 6 stop member to define the first negative configuration and the other of the bearing stop 7 members is engageable with the other of the shaft 8 9 stop members to define the second negative 10 configuration. 11 Preferably, said apparatus is a multi-legged, self-12 levelling space frame equipped with a plurality of 13 14 hydrofoils, typically at different heights. 15 In alternative embodiments, the at least one member 16 is rigidly connected to a support framework and is 17 unsymmetrical. Preferably, the at least one member 18 comprises a disc shaped member which, in use, is 19 adapted to produce positive or negative lift 20 regardless of the direction of flow of fluid 21 22 thereby. Preferably, the disc shaped member 23 produces negative lift. 24 According to a second aspect of the invention, there 25 is provided a method of controlling underwater 26 27 equipment; the method comprising: 28 providing an apparatus having at least one member for generating positive or negative lift; 29 30 attaching the apparatus to underwater 31 equipment;

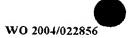
releasing the apparatus into a fluid;





12

1 allowing fluid to flow past the at least one 2 member to generate positive or negative lift. 3 Preferably, the method according to the second 4 aspect of the invention is performed using the 5 6 apparatus according to the first aspect of the 7 invention. 8 9 Preferably, the apparatus is placed in a flow of 10 water. 11 12 Preferably, the underwater equipment is a turbine. 13 14 According to a further aspect of the present invention, there is provided an apparatus for 15 maintaining underwater equipment within a sea or 16 river tidal current location, the apparatus 17 18 comprising at least one moveable members capable of 19 generating negative lift, where said at least one 20 members are moveable between a first configuration 21 in which they create a negative lift due to flow in 22 a first direction, and a second configuration in 23 which they create a negative lift due to flow in a 24 second, different, direction. 25 26 The invention also provides energy extracting apparatus for extracting energy from fluid flow, 27 28 said energy extracting apparatus comprising: 29 a turbine: 30 at least one member, which in use, generates 31 positive or negative lift.



28 29

30



#### PCT/GB2003/003845

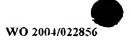
13

1 2 Brief description of the drawings 3 Embodiments of the present invention will now be 4 described, by way of example only, with reference to 5 the accompanying drawings, in which:-6 7 Figure 1 shows a side view of a space frame in 8 accordance with the present invention, showing 9 10 a tubular frame allowing the positioning of the hydrofoils at differing heights; 11 Figures 2a to 2d show the passive reversing of 12 the hydrofoils in response to a change in flow 13 14 direction whilst Figures 2e to 2h show the different movements of hydrofoils of Figure 1 15 actuated by hydraulic motors to create positive 16 and negative lifts during launch, recovery and 17 transitional operations according to the 18 19 present invention; 20 Figures 2i to 2m show the passive reversing of 21 the hydrofoils in response to a change in flow 22 direction; Figure 3 in its upper half shows a first side 23 view, and in its lower half shows an opposite 24 25 side view, illustrating the fundamental geometry of the passive reversing mechanism; 26 Figure 3a in its upper half shows a first side

view, and in its lower half shows an opposite

geometry of the passive reversing mechanism;

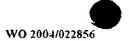
side view, illustrating the fundamental





14

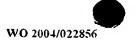
1 Figure 3b is a third side view showing the 2 fundamental geometry of the passive reversing 3 mechanism; 4 Figure 4 shows in detail the assemblage of 5 hydrofoils onto the space frame of Figure 1; 6 Fig. 5a is a side view of a second embodiment 7 of an apparatus in accordance with the present invention and an attached canister; 8 9 Fig. 5b is a front view of the Fig. 5a 10 apparatus with the attached canister; Fig. 5c is a plan view of the Fig. 5a apparatus 11 with the attached canister; and, 12 Figs. 5d-5f are a series of views of an 13 14 attachment ring which forms part of the Fig. 5a 15 apparatus. 16 17 Detailed description of the invention 18 According to the present invention, the apparatus 19 20 for launching an underwater device from a vessel, securing the underwater device whilst in operation 21 22 on the seabed and permitting recovery to a vessel, for maintenance and repair should be as simple as 23 possible without involving any sophisticated and 24 25 specialised equipment. A first embodiment of the invention is shown in Figure 1 and utilises passive, 26 27 self-rectifying static hydrofoils, the central shaft (see Figure 3) of which can be rotated through 180° 28 29 to generate positive or negative lift as required. 30 As is shown in Figure 1, the apparatus 1 for 31 controlling the launch, secure positioning and 32





15

recovery of an underwater device comprises a space 1 2 frame 10 for attaching to any desired underwater device such as power extraction equipment which may 3 comprise a tidal turbine (not shown), a hydrofoil 4 5 support frame to accommodate the self rectifying hydrofoil mechanisms 12 and hydraulically operated 6 legs 11 for levelling of the apparatus 1. The feet 7 14 are equipped with spikes or similar serrated 8 attachments (not shown) to initiate grip on the sea 9 10 or river bed. 11 12 The hydrofoils 12 are inclined in such a way as to generate a significant downforce as a result of the 13 stream flow over their surfaces. This downforce 14 will push the apparatus 1 into the seabed, and, 15 since the actual applied force will be proportional 16 to the square of the velocity of the fluid passing 17 over them, the apparatus 1 will be more securely 18 fixed as the streamflow velocity increases. 19 20 means the apparatus can overcome overturning moments applied to the underwater device that it supports: 21 22 23 The space frame 10 is shown as arched tubing but is 24 not restricted to shape since any frame 25 configuration offering different levels of mounting point for the hydrofoils 12 will suffice. 26 apparatus 1 as shown has multiple hydrofoils 12 but 27 any number of hydrofoils 12 will suffice. As is 28 shown in Figures 2a to 2h, each hydrofoil 12 is 29 mounted on a central shaft 48 such that it may 30 rotate upwards from horizontal (or any angle of 31 inclination above horizontal) through vertical to 32



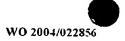
bed.



#### PCT/GB2003/003845

16 any angle above horizontal but now pointing in the 1 opposite direction. The angle of attack of the 3 3 hydrofoils 12 is governed by the relative size and positioning of lugs 46 attached to the central shaft 4 48 and the corresponding lobes 44 attached to an 5 outer shaft (not shown) which is itself fixed to the 6 7 hydrofoil 12. 8 In a preferred embodiment, the apparatus 1 according 9 to the present invention comprises a multi-legged, 10 self-levelling space frame 10 equipped with a number 11 of hydrofoils 12 at different heights with any 12 13 underwater device, such as a tidal turbine, it 14 supports, situated as close as practicable to the 15 centre of gravity of the apparatus 1. 16 17 It is anticipated that the space frame 10 will be mounted on a number of feet 14 equipped with spikes 18 (not shown) to resist slipping of the apparatus 1 19 with respect to the river bed (not shown) by shear 20 force rather than relying on friction alone. 21 number of feet 14A, 14B required will depend on the 22 weight of the apparatus 1; however, the location and 23 24 the shape of these supporting feet 14A, 14B aim at holding the apparatus 1 in the orientation shown in 25 Figure 1 upwards against the current and thus 26 ensuring the stability of the space frame 10. 27 negative lift (arrow A) will tend to force these 28 29 spikes into the sea or river bed (not shown in Figure 1) thus resisting the drag forces acting on 30

the space frame 10 tangentially to the sea or river

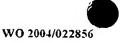


17

1 2 The drag forces acting on the underwater device (such as the tidal turbine) attached to the 3 apparatus 1 will naturally tend to apply an 4 overturning moment to the space frame 10 about its 5 6 rearmost feet 14B, with respect to the direction of flow (arrow F). These forces will however be 7 overcome by positioning the hydrofoils 12 at 8 9 stations such that the negative lift (arrow A), created by the foremost or upstream (those at the 10 left hand side of the space frame 10 as shown in 11 Figure 1) hydrofoils 12 acting over the length of 12 the space frame 10, is arranged to exceed the 13 14 overturning moment. 15 Thus, the space frame 10 is symmetrical about its 16 midpoint M with the hydrofoils 12 being coupled to 17 the space frame 10 in a manner, to be subsequently 18 detailed in a discussion of Figures 2a to 2h, which 19 allows them to passively reverse with stream flow F 20 to maintain compressive forces in a downwards 21 direction A and restraining moments regardless of 22 tidal stream direction. 23 24 During operation of the apparatus 1, the hydrofoils 25 12 are free to rotate (shown as clockwise in Figures 26 2a to 2d and 2I to 2m) in response to the change in 27 tidal stream flow F direction in a manner which is 28 shown from left to right in Figures 2a to 2d to 29 create a negative lift (arrow A) so as to push the 30

apparatus 1 into the seabed.

31



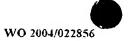


#### PCT/GB2003/003845

18

1 When the apparatus 1 is to be installed on the 2 seabed or is to be recovered from the seabed for e.g. maintenance of the apparatus 1, as shown in the 3 Figures 2a to 2d, hydraulic motors 30, via a 5 suitable gearing mechanism such as a worm and wheel 6 arrangement 32 (as shown in Figure 3) or chain type mechanism (not shown), are utilised to rotate (shown 7 as anticlockwise in Figures 2e to 2h) the 8 9 longitudinal axes (i.e. the horizontal axes 10 perpendicular to the stream flow 12) of the hydrofoils 12 through the required angle until the 11 12 hydrofoils 12 have reached the configuration shown Figure 2h; for the configuration shown in Figures 2e 13 14 to 2h, this angle is approximately 180°. be kept in mind that the hydraulic motors 30 can be 15 16 replaced by pneumatic or electric motors. In other 17 words, if the apparatus 1 is towed, e.g. by a boat 18 or other vessel or installation at the surface, the 19 hydrofoils 12 will produce positive lift (arrow B) 20 as shown in Figures 2e to 2h. For launch and 21 recovery, this positive lift can be utilised to raise or lower the space frame 10 within the tidal 22 23 If required, this action could be augmented by forming air tanks within the space frame 10 that 24 can be 'blown' with compressed air to improve the 25 buoyancy of the apparatus 1. If the hydraulic 26 27 motors 30 use the worm and wheel mechanism 32 form 28 of drive, the hydrofoil 12 positions can be altered over a range of positions, thus permitting the 29 apparatus 1 to be 'flown' in the water. Hydraulic

connections (and pneumatic connections if required)





19

- can be affixed to a supporting marker buoy (not
- 2 shown) for ease of access.

3

- Figure 3 shows the mechanism and assemblage of
- 5 hydrofoils 12, hydraulic motors 30 and worm and
- 6 wheel drive shaft mechanisms 32 in more detail. The
- 7 hydrofoils 12 are free to rotate about a central
- 8 shaft 48, through an included angle of say 160°
- 9 which will maintain an angle of 10° to the
- 10 horizontal. The 10° angle effectively becomes an
- 11 angle of attack when the tidal stream flow F
- 12 reverses. Thus as the tidal stream 10 reciprocates,
- 13 the hydrofoils 12 will maintain an angle of 10°,
- 14 creating a negative lift (arrow A), which will
- therefore push the spikes 16 into the seabed and
- 16 immobilise the space frame 10. As will be described
- 17 subsequently, positioning lugs 46 mounted on a
- 18 central shaft 48 provided a stop for locating lobes
- 19 44 of the hydrofoil 12, such that the hydrofoil 12
- 20 cannot rotate further than the 160° shown in Figures
- 21 2a to 2d.

22

- 23 By rotating the central shaft 48 through slightly
- greater than 180° (say 200°), the negative lift
- 25 becomes positive lift (arrow B) and the space frame
- 26 10 will rise through the water so that the tidal
- 27 turbine 90 can be recovered on the vessel (not
- 28 shown).

- 30 Figure 4 shows in more detail the mechanical
- 31 assemblage of hydrofoils 12 with space frame 10.
- 32 The hydraulic motor 30 for actuating the positioning

WO 2004/022856



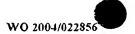
## PCT/GB2003/003845

20

1 gear is equipped with a drive shaft 32 that is utilised for rotating an indented positioning gear 2 42 or a toothed gear wheel. 3 The positioning gear 42 is solidly attached to a central shaft 48 which 4 passes through a bore provided in the larger end of 5 6 each hydrofoil 12, a section of which is show on Figure 4. The bore of the hydrofoil 12 is provided 7 8 with a pair of diametrically opposed and inwardly 9 projecting hydrofoil locating lobes 44. The central shaft 48 has a pair of diametrically opposed and 10 outwardly projecting positioning lugs 46, each one 11 12 of which selectively co-operates with one of the respective pair of diametrically opposed hydrofoil 13

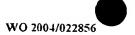
14 15 locating lobes 44.

16 Thus, by rotating the drive shaft 32, the hydraulic motor 30 actuates or rotates the position gear 42 17 18 which in turn rotates the central shaft 48. positioning lugs 46 will contact the locating lobes 19 44 and carry them 44 (and the hydrofoil 12) about 20 the rotational axis of the central shaft 48 until 21 the hydrofoil 12 is in the desired configuration, 22 23 this being through an angle of approximately 160° until the hydrofoil 12 is in the configuration shown 24 25 in Figure 2h. At this point, the motor 30 is deactuated and the positioning lugs 46 will hold the 26 27 hydrofoil 12 locked in this configuration. 28 rotation of 160° enables the hydrofoil 12 to maintain an angle of 10° to the horizontal in order 29 to provide an angle of attack when the tidal stream 30 31 F reverses.





Conversely, the rotation of the central shaft 48 by 1 180° drives the hydrofoils 12 to create a positive 2 lift and in which case, the space frame 10 will rise 3 through water. Figure 3a shows how the attitude of 4 the hydrofoil 12 is changed by a simple 180° 5 6 clockwise rotation of the central shaft 48. 7 The apparatus according to the present invention, 8 can be launched and recovered by a non-specialist 9 vessel, using non-specialist equipment. Indeed if 10 the vessel is large enough, a number of apparatus 1 11 may be launched or recovered in a day without the 12 need to return to port. This will also permit easy 13 access for maintenance and repair. Since apparatus 14 15 1 possesses few moving parts and no complex mechanisms, it should be inherently reliable. 16 17 A second embodiment of an apparatus in accordance 18 19 with the present invention is shown in Figs. 5a-5d. The apparatus 100 comprises a tripod support frame 20 21 110, a bottom ring or stand 126, a disc-shaped hydrofoil 112, support brackets 120 and an 22 attachment ring 122 with bolts 123. The apparatus 23 24 100 is attached to an ADCP canister 124 via the attachment ring 122 and bolts 123. Other subsea 25 26 equipment may also be attached to the apparatus 100 in place of the canister 124. 27 28 The hydrofoil 112 is rigidly connected to the frame 29 110 via the support brackets 120 and its plane is 30 generally parallel to the main plane defined by the 31 32 bottom ring 126 such that the hydrofoil 122 will be





22

- generally parallel to the seabed in use. A central
- 2 aperture 119 is provided within the hydrofoil 112.
- 3 A lower face 113 of the hydrofoil 112 faces the
- 4 stand 126 and is of a generally flat surface,
- 5 whereas its opposite, upper, face 115 faces away
- from the stand 126 and gradually curves upwards away
- 7 from the main plane of the hydrofoil as it
- 8 approaches the central aperture 119 to form a raised
- 9 lip portion 117. This can be achieved by the
- assembly of a plurality of smaller hydrofoils 112s
- 11 to produce a multi-faceted hydrofoil 112. The
- 12 hydrofoil 112 thus has rotational symmetry around a
- 13 central axis 118 but is not symmetrical on either
- 14 side of its main plane.

15

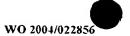
- 16 Thus when a flow of water passes over each face 113,
- 17 115 of the hydrofoil 112, the reaction force of the
- water on the raised lip 117 pushes the hydrofoil 112
- 19 along with the other components of the apparatus 100
- 20 and ADCP canister 124 in a downwards direction -
- 21 that is "negative lift" results.

22

- 23 Thus in use, the hydrofoil helps to direct the
- 24 apparatus 100 and attached equipment towards the
- 25 seabed and once in position, the hydrofoil maintains
- the apparatus and equipment on the seabed.

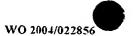
27

- 28 The apparatus 100 may be attached to a line (not
- shown) and the line attached at its other end to a
- 30 buoy. If the apparatus needs to be recovered, the
- 31 apparatus may be pulled in by the line.





An advantage of certain embodiments of the present 1 invention, such as the second embodiment. is that 2 they continue to perform their function of providing 3 negative lift regardless of the direction of flow of 4 5 the water. 6 An advantage of the second embodiment of the 7 invention is that it includes no moving parts and so 8 is reliable and requires minimal maintenance. 9 10 The embodiments described herein may also be 11 provided with an integral turbine or other 12 13 underwater equipment rather than attaching such 14 equipment to the apparatus before use. 15 Although reference is made to employing the 16 apparatus 1, 100 in a tidal current and in certain 17 embodiments using a tidal turbine, it is to be 18 understood that the apparatus 1, 100 may be placed 19 in any flow of liquid such as rivers and are not 20 21 limited to their use tidal areas. 22 An advantage of certain embodiments of the present 23 24 invention is that they permit the launch and recovery of underwater equipment to be carried out 25 26 using a non-specialist but suitably equipped vessel. 27 Concerning the primary environmental impact of 28 embodiments of apparatus 1 according to the present 29 30 invention, it would have some impact upon the benthic flora and fauna, and, although the 31 positioning and retrieval of apparatus 1 would be 32

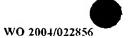




24

relatively frequent (at least once every year is 1 anticipated), nothing more than temporary localised 2 disturbance is anticipated. There exists some 3 potential for hydraulic oil leakage, but the system 4 contents are minimal so, even in the event of 5 complete system evacuation, any oil contamination 6 would be minor. Operational environmental hazards 7 are in common with the other forms of tidal energy 8 9 extraction and decommissioning would leave no 10 footprint. 11 12 Improvements and modifications in terms of dimensions and locations of the different parts 13 14 described above may be incorporated to the hereinbefore described apparatus for controlling the 15 launch and recovery of a tidal turbine without 16

departing from the scope of the present invention.



25

1 CLAIMS:-

2

- 3 1. Apparatus for controlling underwater equipment
- 4 comprising:
- 5 attachment means for attaching underwater
- 6 equipment to the apparatus; and
- 7 at least one member for generating positive or
- 8 negative lift.

9

- 10 2. Apparatus according to claim 1, wherein the at
- least one member is adapted to create a negative
- 12 lift due to fluid flow in a first direction and is
- 13 adapted to create a negative lift due to fluid flow
- in a second, different, direction.

15

- 16 3. Apparatus as claimed in claim 2, wherein the
- 17 first and second directions are generally opposite
- 18 to each other.

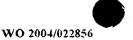
19

- 4. Apparatus as claimed in any preceding claim,
- 21 which, in use, is adapted to anchor the underwater
- 22 equipment to a sea- or river-bed.

23

- 24 5. Apparatus according to any preceding claim,
- 25 wherein the attachment means is adapted to attach
- the underwater equipment in close proximity to the
- 27 centre of gravity of the apparatus.

- 29 6. Apparatus according to any preceding claim,
- 30 wherein the apparatus is mounted on a number of feet
- 31 equipped with slippage prevention means, to resist
- 32 slipping by shear force such that, in use, the



26

- 1 negative lift will preferably tend to force said
- 2 slippage prevention means into a sea- or river-bed
- 3 thus resisting the drag forces acting on the
- 4 apparatus tangentially to the seabed.

5

- 6 7. Apparatus as claimed in any preceding claim,
- 7 wherein the at least one member comprises at least
- 8 one hydrofoil.

9

- 10 8. Apparatus according to any preceding claim,
- wherein differences in pressure acting on opposing
- 12 surfaces of the at least one member due to a
- 13 predetermined angle of attack causes said at least
- one member to generate negative or positive lift.

15

- 16 9. Apparatus as claimed in any preceding claim,
- 17 which is adapted to control the launch and/or
- 18 recovery of the underwater equipment.

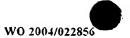
19

- 20 10. Apparatus according to any preceding claim,
- 21 wherein the at least one member is free to rotate
- 22 through a pre-determined angle.

23

- 24 11. Apparatus according to any preceding claim,
- 25 wherein the at least one member comprises at least
- one hydrofoil capable of passive rotation about an
- 27 axis such that each hydrofoil maintains alignment
- with a periodically reciprocating rectilinear flow.

- 30 12. Apparatus as claimed in any preceding claim,
- 31 wherein said at least one member is moveable between
- 32 a first configuration in which it is capable of



27

- generating positive lift and a second configuration
- in which it is capable of generating negative lift.

3

- 4 13. Apparatus according to claim 12, wherein the at
- 5 least one member has a variable actuating means to
- 6 vary the positive or negative lift generated by the
- 7 member.

8

- 9 14. Apparatus according to claim 12 or 13, wherein
- the at least one member is rotatable between said
- 11 first and second configurations about a longitudinal
- 12 axis thereof.

13

- 14 15. Apparatus according to any one of claims 12-14,
- wherein a shaft member is adapted to actuate the at
- least one member to change it between the first and
- 17 second configurations.

18

- 19 16. Apparatus according to any one of claims 10-15
- 20 further comprising a support framework, where a
- 21 plurality of shaft members are connected to the
- 22 framework and on which said at least one member is
- 23 rotatably coupled.

24

- 25 17. Apparatus according to claim 16, wherein the at
- least one member comprises a bearing member by means
- of which it is coupled to a shaft member connected
- 28 to the support framework.

- 30 18. Apparatus according to claim 17, wherein the
- 31 bearing member and shaft member combine to provide a



WO 2004/022856

#### PCT/GB2003/003845

28

- rotation enabling portion and a rotation prevention
- 2 portion.

3

- 4 19. Apparatus according to claim 18, wherein the
- 5 rotation prevention portion comprises one or more
- stop members which are adapted to engage with one or
- 7 more respective stop members mounted on the
- 8 respective bearing member.

9

- 10 20. Apparatus according to claim 19, wherein the
- 11 bearing member is substantially cylindrical and
- 12 comprises a pair of stop members which are spaced
- 13 apart around its inner circumference.

14

- 15 21. Apparatus according to claim 19 or 20, wherein
- 16 the shaft member comprises a pair of stop members
- 17 which are spaced apart around its outer
- 18 circumference.

19

- 20 22. Apparatus according to claim 21, wherein one of
- 21 the bearing stop members is engageable with a
- 22 respective shaft stop member to define a first
- 23 negative configuration and the other of the bearing
- 24 stop members is engageable with the other of the
- 25 shaft stop members to define a second negative
- 26 configuration.

27

- 28 23. Apparatus as claimed in any one of claims 1-8,
- 29 wherein the at least one member is rigidly connected
- 30 to a support framework.

## 3.0 ம்.த

29

- 1 24. Apparatus as claimed in claim 23, wherein the
- 2 at least one member comprises a disc shaped member
- 3 which, in use, is adapted to produce positive or
- 4 negative lift regardless of the direction of flow of
- 5 fluid thereby.

€.

- 7 25. Energy extracting apparatus for extracting
- 8 energy from fluid flow, said energy extracting
- 9 apparatus comprising:
- 10 a turbine;
- at least one member, which in use, generates
- 12 positive or negative lift.

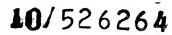
13

- 14 26. A method of controlling underwater equipment;
- 15 the method comprising:
- 16 providing an apparatus having at least one
- 17 member for generating positive or negative lift;
- 18 attaching the apparatus to underwater
- 19 equipment;
- 20 releasing the apparatus into a fluid;
- allowing fluid to flow past the at least one
- 22 member to generate positive or negative lift.

23

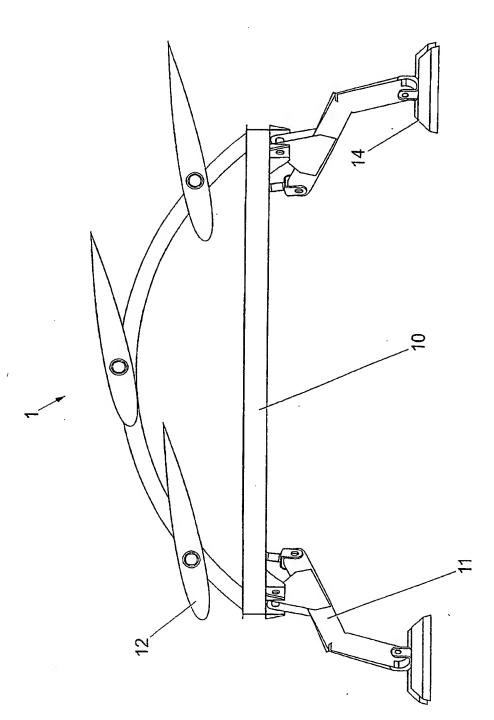
- 24 27. A method as claimed in claim 26, wherein the
- 25 apparatus is placed in a flow of water.

- 27 28. A method as claimed in claim 26, wherein the
- 28 underwater equipment is a turbine.





1/8



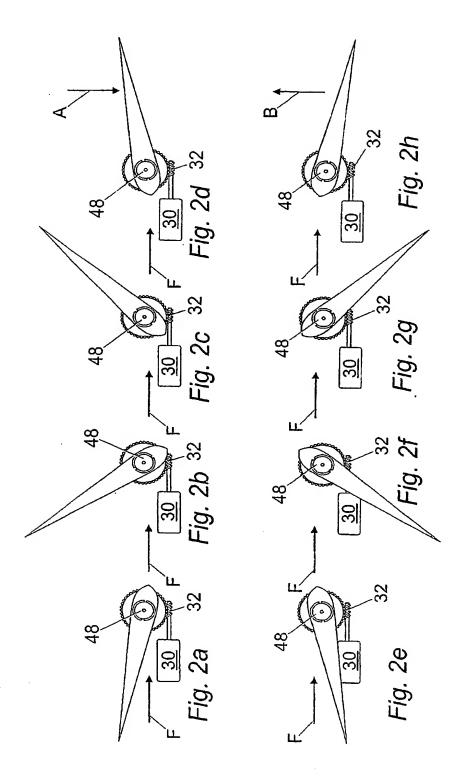
⊢/g. 7

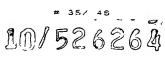


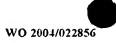
WO 2004/022856

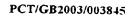
PCT/GB2003/003845

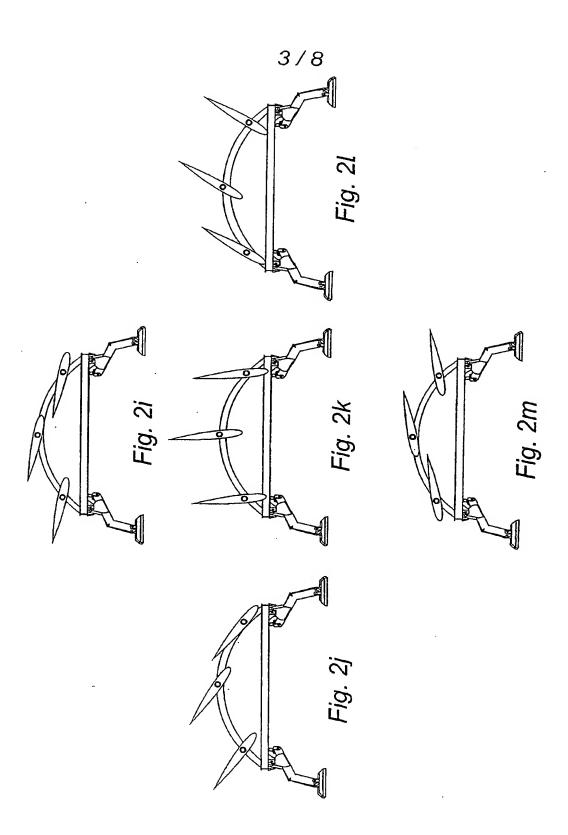
2/8



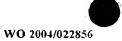


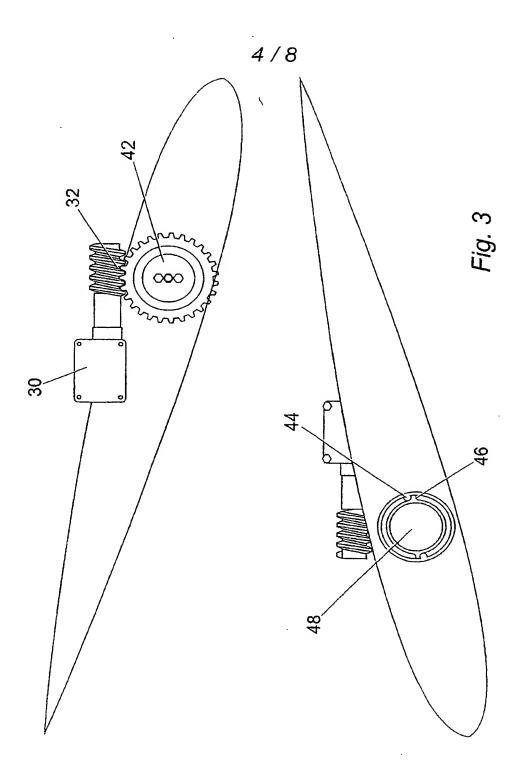








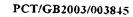




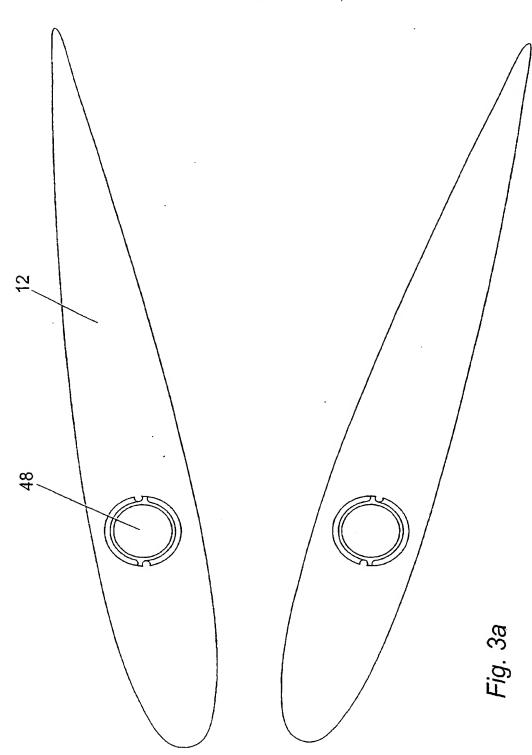
10/526264

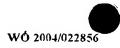






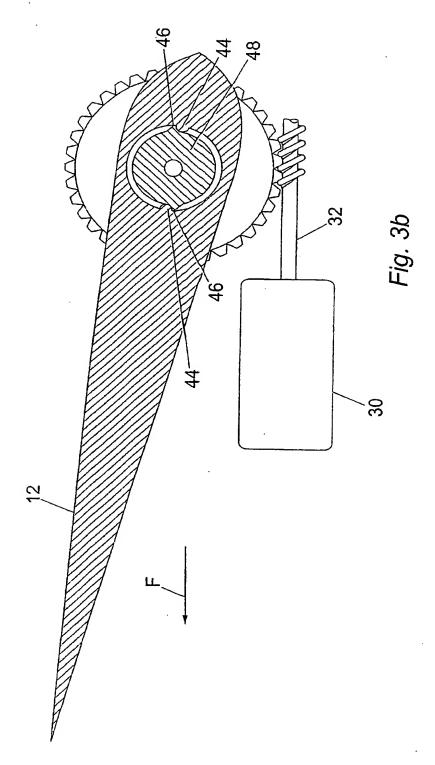












WO 2004/022856

PCT/GB2003/003845

7/8

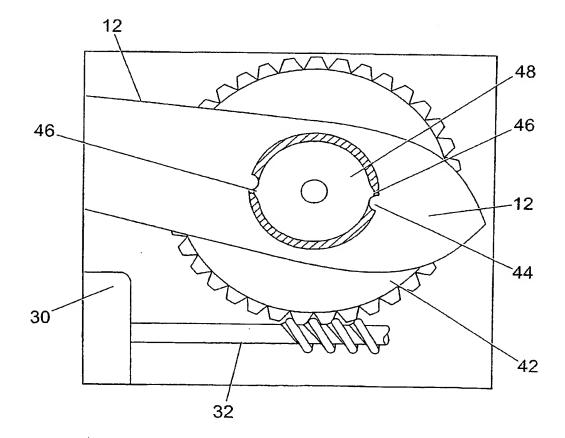
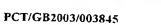
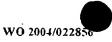
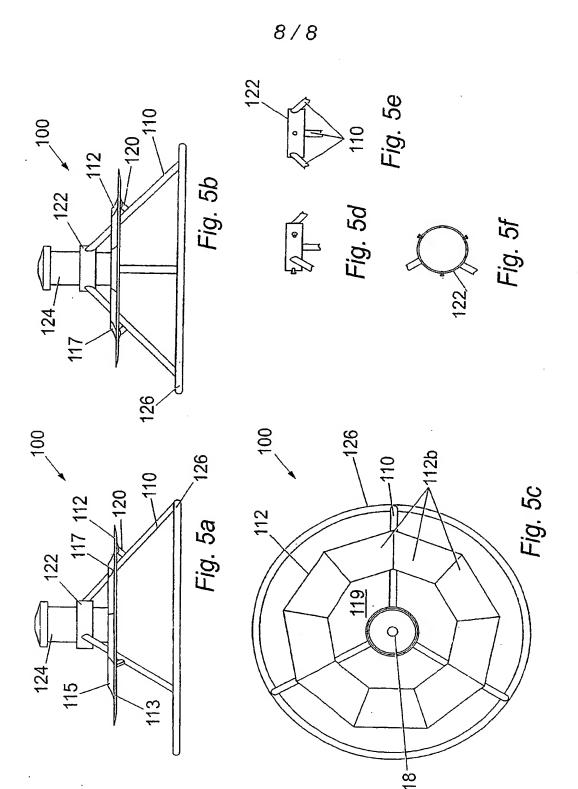


Fig. 4













Internation INTERNATIONAL SEARCH REPORT cation No PCT/GB U3/03845 A. CLASSIFICATION OF SUBJECT MATTER IPC 7 E0289/08 According to International Patent Classification (IPC) or to both national classification and IPC B. FIELDS SEARCHED Minimum documentation searched (classification system followed by classification symbols) IPC 7 E02B Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched Electronic data base consulted during the international search (name of data base and, where practical, search terms used) EPO-Internal, WPI Data C. DOCUMENTS CONSIDERED TO BE RELEVANT Citation of document, with indication, where appropriate, of the relevant passages Relevant to claim No. χ US 3 978 345 A (BAILEY DAVID ZABRISKIE) 1-4 31 August 1976 (1976-08-31) 7-14,23, 25-28 column 2, line 12 -column 5, line 23; figures 6,7 χ DE 199 58 409 A (KIMMIG PETER) 1,4-6,9 13 June 2001 (2001-06-13) 12,23,25 the whole document P,X US 2003/072615 A1 (GARICA FELIPE ET AL) 1 17 April 2003 (2003-04-17) paragraph '0052! - paragraph '0053!; figure 7 Α US 4 086 775 A (PETERSON JR CHARLES A) 2 May 1978 (1978-05-02) Further documents are listed in the continuation of box C. Patent family members are listed in annex. ΙX Special categories of cited documents: \*T\* later document published after the international filing date or priority date and not in conflict with the application but clied-to understand the principle or theory underlying the \*A\* document defining the general state of the art which is not considered to be of particular relevance 'E' earlier document but published on or after the international \*X\* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person stilled in the art. 'O' document referring to an oral disclosure, use, exhibition or \*P\* document published prior to the international filing date but fater than the priority date claimed "&" document member of the same patent family Date of the actual completion of the international search Date of malling of the international search report

19/01/2004

Flygare, E

Authorized officer

Farm PCT/ISA/213 (second sheel) (July 1992)

Name and mailing address of the ISA

8 January 2004

European Palent Office, P.B. 5818 Palentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Tx. 31 651 epo nl, Fax: (+31-70) 340-3016





information on patent family members

Internatio:	lication No
PCT/GD	us/03845

Palant days		~			101740 03/03845	
Patent document cited in search report		Publication date		Patent family member(s)	Publication date	
US 3978345	A	31-08-1976	AU CA DE GB JP JP JP NL	8139975 Å 1062584 A1 2523030 A1 1515561 A 1373688 C 50155841 A 61030154 B 7506125 A ,B,	25-11-1976 18-09-1979 04-12-1975 28-06-1978 07-04-1987 16-12-1975 11-07-1986 26-11-1975	
DE 19958409.	Α	13-06-2001	DE	19958409 A1	13-06-2001	
US 2003072615	A1	17-04-2003	NONE			
US 4086775	A	02-05-1978	NONE			

Form PCT/ISA/210 (patent family arnex) (July 1992)